### Robust Control – Final Project

# µ-synthesis Design of A Half-Car Active Suspension System

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### Outlines







Half-Car Model Passenger Model Hydraulic Actuator



### Results

# μ-synthesis



### Structured Singular Value



Figure 17.4 System with structured uncertainty

**Definition 17.1** For any given matrix  $M \in \mathbb{C}^{n \times n}$ , the structured singular value  $\mu_{\Delta}(M)$  is defined as

$$\mu_{\Delta}(M) = \frac{1}{\min\{\sigma_{\max}(\Delta) \mid \Delta \in \Delta, \ \det(I - M\Delta) = 0\}}.$$
(17.10)  
(17.10) when there is no  $\Delta \in \Delta$  satisfying dot $(I - M\Delta) = 0$ 

 $\mu_{\Delta}(M) = 0$  when there is no  $\Delta \in \Delta$  satisfying  $\det(I - M\Delta) = 0$ .



### Structured Singular Value



Figure 17.4 System with structured uncertainty

### 17.3.1.1 Single Scalar Block Uncertainty $\Delta = \{ \delta I | \delta \in \mathbb{C} \}$

In this case,  $\mu_{\Delta}(M) = \rho(M)$  holds. Here,  $\rho(M)$  denotes the spectral radius of matrix M, that is, the maximum absolute value of all eigenvalues of M.

*Proof.* First,  $\det(I - M\delta) = \det(\delta^{-1}I - M) \det(\delta I) = 0$  holds. So any nonzero  $\delta^{-1}$  satisfying this equation is an eigenvalue of M. Then, it is easy to see that the reciprocal of the minimum size of uncertainty  $\delta$  satisfying this equation is the spectral radius  $\rho(M)$  of matrix M, that is,

$$\mu_{\Delta}(M) = \frac{1}{\min(|\delta| : \det(I - M\delta) = 0)}$$

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## Robust Stability and Robust Performance



Figure 8.2:  $N\Delta$ -structure for robust performance analysis

- NS  $\Leftrightarrow$  N (internally) stable
- NP  $\Leftrightarrow \bar{\sigma}(N_{22}) = \mu_{\Delta_P} < 1, \forall \omega, \text{ and NS}$
- RS  $\Leftrightarrow \mu_{\Delta}(N_{11}) < 1, \forall \omega, \text{ and NS}$

$$\operatorname{RP} \quad \Leftrightarrow \quad \mu_{\widetilde{\Delta}}(N) < 1, \; \forall \omega, \; \widetilde{\Delta} = \begin{bmatrix} \Delta & 0 \\ 0 & \Delta_P \end{bmatrix}, \text{ and } \operatorname{NS}$$



### $\boldsymbol{\mu}$ synthesis and DK-iteration

- ▶  $\max \rho(QN) \leq \mu(N) \leq \inf \overline{\sigma}(DND^{-1})$
- $\min_{K} (\min_{D \in \mathcal{D}} \|DN(K)D^{-1}\|_{\infty})$



### **DK-iteration**

- 1. **K-step.** Synthesize an  $\mathcal{H}_{\infty}$  controller for the scaled problem,  $\min_{K} ||DN(K)D^{-1}||_{\infty}$  with fixed D(s).
- 2. **D-step.** Find  $D(j\omega)$  to minimize at each frequency  $\bar{\sigma}(DND^{-1}(j\omega))$  with fixed N.
- 3. Fit the magnitude of each element of  $D(j\omega)$  to a stable and minimum phase transfer function D(s) and go to Step 1.

# Problem Statement

**Ride Comfort and Suspension Deflection** 







A Half-Car Active Suspension System



Ride Comfort and Suspension Deflection?



A Half-Car Active Suspension System



- ▶ In general,
  - $\ddot{X_s}$  represents ride comfort of a vehicle.





- ▷  $(X_{sf} X_{uf})$  represents the suspension deflection.
- ▷  $(X_{sr} X_{ur})$  represents the suspension deflection.



# System Structure

Suspension System + Passenger Model + Hydraulic System



### Overall Structure

### A Half-Car Active Suspension System













Fig. 1. Half car 4 DOF active suspension model









Fig. 1. Half car 4 DOF active suspension model



Equations of motion





$$m_{s}\ddot{X}_{s} + c_{sf}(\dot{X}_{sf} - \dot{X}_{uf}) + k_{sf}(X_{sf} - X_{uf}) + c_{sr}(\dot{X}_{sr} - \dot{X}_{ur}) + k_{sr}(X_{sr} - X_{ur}) - f_{sf} - f_{sr} = 0$$
(1)  

$$I_{s}\ddot{\Theta}_{s} + L_{f}\left[c_{sf}(\dot{X}_{sf} - \dot{X}_{uf}) + k_{sf}(X_{sf} - X_{uf}) - f_{sf}\right] - L_{r}\left[c_{sr}(\dot{X}_{sr} - \dot{X}_{ur}) + k_{sr}(X_{sr} - X_{ur}) - f_{sr}\right] = 0$$
(2)  

$$m_{uf}X_{uf} - c_{sf}(X_{sf} - X_{uf}) - k_{sf}(X_{sf} - X_{uf}) + k_{uf}(X_{uf} - w_{sf}) + f_{sf} = 0$$
(3)

$$m_{ur}\ddot{X}_{ur} - c_{sr}(\dot{X}_{sr} - \dot{X}_{ur}) - k_{sr}(X_{sr} - X_{ur}) + k_{ur}(X_{ur} - w_{sr}) + f_{sr} = 0$$
(4)

Constraints

$$X_{s} = (L_{f}X_{sr} + L_{r}X_{sf}) / L$$
  

$$\Theta_{s} = (X_{sf} - X_{sr}) / L$$



- $\flat \quad \dot{x} = Ax + Bu$
- ▷ y = Cx + Du



Fig. 1. Half car 4 DOF active suspension model

$$x = \begin{bmatrix} \dot{x}_{sf} & \dot{x}_{uf} & \dot{x}_{sr} & \dot{x}_{ur} & x_{sf} & x_{uf} & x_{sr} & x_{ur} \end{bmatrix}^{T}$$
$$u = \begin{bmatrix} w_{sf} & w_{sr} & f_{sf} & f_{sr} \end{bmatrix}^{T}$$







Uncertain Biodynamics







Equations of motion

$$m_{H}\ddot{z}_{H} = -k_{H-UT}\left(z_{H} - z_{UT}\right) - c_{H-UT}\left(\dot{z}_{H} - \dot{z}_{UT}\right)$$
(1)

$$m_{UT}\ddot{z}_{UT} = k_{H-UT} \left( z_H - z_{UT} \right) - k_{UT-LT} \left( z_{UT} - z_{LT} \right) - k_{UT-T} \left( z_{UT} - z_T \right) + c_{H-UT} \left( \dot{z}_H - \dot{z}_{UT} \right) - c_{UT-LT} \left( \dot{z}_{UT} - \dot{z}_{LT} \right) - c_{UT-T} \left( \dot{z}_{UT} - \dot{z}_T \right)$$
(2)

$$m_{LT} \ddot{z}_{LT} = k_{UT-LT} \left( z_{UT} - z_{LT} \right) - k_{LT-T} \left( z_{LT} - z_{T} \right) + c_{UT-LT} \left( \dot{z}_{UT} - \dot{z}_{LT} \right) - c_{LT-T} \left( \dot{z}_{LT} - \dot{z}_{T} \right)$$
(3)

$$m_{T}\ddot{z}_{T} = k_{UT-T} \left( z_{UT} - z_{T} \right) + k_{LT-T} \left( z_{LT} - z_{T} \right) - k_{T-se} \left( z_{T} - z_{se} \right) + c_{UT-T} \left( \dot{z}_{UT} - \dot{z}_{T} \right) + c_{LT-T} \left( \dot{z}_{LT} - \dot{z}_{T} \right) - c_{T-se} \left( \dot{z}_{T} - \dot{z}_{se} \right)$$
(4)

$$m_{se} \ddot{z}_{se} = k_{T-se} \left( z_T - z_{se} \right) - k_{se} \left( z_{se} - z_p \right) + c_{T-se} \left( \dot{z}_T - \dot{z}_{se} \right) - c_{se} \left( \dot{z}_{se} - \dot{z}_p \right)$$
(5)





Xsr 1

Xur

Wsr







 $z_p = x_s + P_x \,\theta_s$ 





Figure 2: Hydraulic actuator block diagram



▶ Oil waft from the pump,  $q_p = K_p \frac{dx}{dt}$ 





- ▶ Oil glide via the motor,  $q_m = K_m \frac{d\theta}{dt}$
- $q_i = K_i P$ Leakage flow rate, ⊳
- ▶ Compressibility flow rate,  $q_c = K_c \frac{dp}{dt}$



▶ Oil waft from the pump,  $q_p = K_p \frac{dx}{dt}$ 





- ▶ Oil glide via the motor,  $q_m = K_m \frac{d\theta}{dt}$
- $q_i = K_i P$ Leakage flow rate, ⊳
- Compressibility flow rate, ⊳

$$q_c = K_c \frac{dp}{dt}$$

$$K_{p}\frac{dx}{dt} = K_{m}\frac{d\theta}{dt} + K_{i}P + K_{c}\frac{dP}{dt}$$



- Assumptions:
  - ► Km = Kt = Kc
  - ► Tm = TI
  - ► Kc = 0
- Transfer function

$$\frac{\theta(s)}{X(s)} = \frac{K_p}{\left[\frac{K_i J}{K_m}s + \frac{K_m^2 + K_i B}{K_m}\right]}$$





⊳



Chain

Output Angular Displacment Ø

Motor

▶ purturbed 
$$\widetilde{G_1(s)} = \frac{K_1}{T_1s+1}$$
 with multiplicative uncertainty

• 
$$\widetilde{G_1} = G_1 (1 + W_{m1}\Delta_1)$$
, where  $\|\Delta_1\|_{\infty} < 1$ .

- $K_1$ : uncertainty of 10%
- $T_1$ : uncertainty of 20%









#### Car Chassis Hydraulic Actuator 0.3803s + 60.8973 Low Pressure Line Pump $W_{m1} =$ Motor Output Angular Displacment B *s* **+ 599.5829** High Pressure Line Wheel Assemb Approximation of the first actuator transfer function nt X -5 actuator block diagram -10 -15 Magnitude (dB) -20 -25 -30 $| (G1(j\omega) - G1nom(j\omega)) / G1nom(j\omega) |$ -35 Wm1(jω) | -40 10<sup>2</sup> $10^{3}$ 10<sup>4</sup> 10<sup>1</sup> Frequency (rad/s)

Chair



#### Car Chassis Hydraulic Actuator Chair Low Pressure 0.3803s + 60.8973 Line $W_{m1} =$ Motor Output Angular Displacment @ *s* + 599.5829 High Pressure Line Wheel Assemb Approximation of the first actuator transfer function -5 actuator block diagram -10 -15 Magnitude (dB) -20 $G_1(s) = G_1(s)(1 + W_{m1}\Delta_1)$ -25 -30 $| (G1(j\omega) - G1nom(j\omega)) / G1nom(j\omega) |$ -35 Wm1(jω) | -40 10<sup>2</sup> $10^{3}$ $10^{4}$ 10<sup>1</sup> Frequency (rad/s)



From modeling to  $\mu$  -synthesis design





## State-Space

- Plant P
  - $\dot{x} = Ax + Bu$
  - y = Cx + Du







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## State-Space

•  $\dot{x} = Ax + Bu$ 

• 
$$y = Cx + Du$$

$$u^{T} = [w_{sf}, w_{sr}, f_{sf}, f_{sr}]$$

$$y^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur}), \ddot{z_{H}}]^{x_{sr}} \xrightarrow{\text{Ims, Is}} \xrightarrow{I$$

hcar.StateName

v\_sf

2 v\_uf 3 v\_sr 4 v\_ur 5 x\_sf 6 x\_uf

7 x\_sr 8 x\_ur

9 Z\_H 10 Z\_UT 11 Z\_LT 12 Z\_T 13 Z\_se 14 Z\_H\_dot 15 Z\_UT\_dot 16 Z\_LT\_dot

x =



## State-Space

- $\dot{x} = Ax + Bu$
- y = Cx + Du









## State-Space





Xsr 1

Xur

Wsr

ksr

$$u^{T} = [w_{sf}, w_{sr}, f_{sf}, f_{sr}]$$
  

$$y^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur}), \dot{z_{H}}]$$
  

$$z^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur})]$$

 $v^T = [\ddot{z_H}]$ 





### Uncertainties





Uncertain continuous-time state-space model with 3 outputs, 4 inputs, 22 states. The model uncertainty consists of the following blocks:

C\_H\_UT: Uncertain real, nominal = 310, variability = [-15,15]%, 1 occurrences C\_LT\_T: Uncertain real, nominal = 330, variability = [-15,15]%, 1 occurrences C\_T\_se: Uncertain real, nominal = 2.48e+03, variability = [-15,15]%, 1 occurrences C\_UT\_LT: Uncertain real, nominal = 200, variability = [-15,15]%, 1 occurrences C\_UT\_T: Uncertain real, nominal = 909, variability = [-15,15]%, 1 occurrences C\_se: Uncertain real, nominal = 150, variability = [-15,15]%, 1 occurrences Delta\_act1: Uncertain 1x1 LTI, peak gain = 1, 1 occurrences Delta\_act2: Uncertain 1x1 LTI, peak gain = 1, 1 occurrences K1: Uncertain real, nominal = 1.08, variability = [-10,10]%, 1 occurrences K2: Uncertain real, nominal = 1.08, variability = [-10,10]%, 1 occurrences T1: Uncertain real, nominal = 0.005, variability = [-20,20]%, 1 occurrences



### Uncertainties





Uncerta The mo	Dampers in the Passenger Model	3 outputs, 4 inputs, 22 states. .ks <sup>.</sup>
C_H_U	JT: Uncertain real, nominal = 310, variability	y = [-15,15]%, 1 occurrences
C_LT_	T: Uncertain real, nominal = 330, variability	y = [-15,15]%, 1 occurrences
C_T_s	e: Uncertain real, nominal = 2.48e+03, vari	iability = [-15,15]%, 1 occurrences
C_UT_	LT: Uncertain real, nominal = 200, variability	ity = [-15,15]%, 1 occurrences
C_UT_	T: Uncertain real, nominal = 909, variability =	y = [-15,15]%, 1 occurrences
Delta_a	act1: Uncertain 1x1 L1I, peak gain = 1, 1 o	ccurrences
Delta_a	act2: Uncertain 1x1 LTI, peak gain = 1, 1 o	ccurrences
K1: Un	certain real, nominal = 1.08, variability = [-1	0,10]%, 1 occurrences
K2: Un	certain real, nominal = 1.08, variability = [-1	0,10]%, 1 occurrences
T1: Un	certain real, nominal = 0.005, variability = [-	-20,20]%, 1 occurrences
T2: Un	certain real, nominal = 0.005, variability = [-	-20,20]%, 1 occurrences

**Uncertainties in Two Hydraulic Actuators** 









Robust performance			Fit order	
Iter	K Step	Peak MU	D Fit	D
1	2.46	0.2235	0.2278	42
2	0.2278	0.2256	0.2279	58
3	0.2279	0.2256	0.2278	56



 $\mu$  synthesis – Part 1

From  $w_{sf}$ 









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 $u_{\Delta}$ 

w

u

### $\mu$ synthesis – Part 2

v

Δ

P

K







### Robust Performance



















$$z^{T} = \left[ \left( x_{sf} - x_{uf} \right), \left( x_{sr} - x_{ur} \right) \right]$$
$$v^{T} = \left[ \ddot{x_{s}}, \ \ddot{Z_{H}} \right]$$

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### μ synthesis – Part 3

### D-K ITERATION SUMMARY:

Robust performance				Fit orde	
Iter	K Step	Peak MU	D Fit	D	
1	2.46	0.2235	0.2277	40	
2	0.2277	0.2256	0.2277	UZ	
3	0.2277	0.2256	0.2278	50	
Deete					

Best achieved robust performance: 0.223

$$z^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur})]$$
  
$$v^{T} = [\ddot{x_{s}}, \ddot{Z_{H}}]$$



 $\mu$  synthesis – Part 3 From  $W_{sf}$ 

From  $W_{sr}$ 







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$$z^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur}), \ddot{x_{s}}, \ddot{Z_{H}}]$$
$$v^{T} = [\ddot{x_{s}}, \ddot{Z_{H}}]$$



### D-K ITERATION SUMMARY:

Robust performance				Fit order	
Iter	K Step	Peak Ml	J D Fit	D	
1	26.1	25.8	26.04	8	
2	26.04	25.8	26.09	20	
3	26.09	25.8	26.09	12	
Best achieved robust performance: 25.8					



$$z^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur}), \ddot{x}_{s}, \ddot{Z}_{H}]$$
$$v^{T} = [\ddot{x}_{s}, \ddot{Z}_{H}]$$









 $\mu$  synthesis – Part 4 (samples = 100)







### $\mu$ synthesis – Best Case

$$W_1(s) = \frac{0.72s}{s^2 + 7.2s + 5200}$$

(band-pass filter)





### μ synthesis – Best Case





### μ synthesis – Best Case

D-K ITERATION SUMMARY:

 $v^{T} = [(x_{sf} - x_{uf}), (x_{sr} - x_{ur})]$ 

	Robust performance			
lter	K Step	Peak Ml	J D Fit	D
1	2.46	0.301	0.3036	66
2	0.3036	0.301	0.3026	62
3	0.3026	0.301	0.3024	62

Best achieved robust performance: 0.301





### $\mu$ synthesis – Best Case



 $\Delta$ 

P

 $u_{\Delta}$ 

 $y_{\Delta}$ 



### Conclusion

- achieves robust performance
- improves ride comfort
- improves suspension deflections



### Conclusion

 This project establishes a robust feedback control synthesis for a class of half-car suspension systems considering a 4-DOF passenger's biodynamics with parametric uncertainties.



### Reference

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- 2. Gudarzi M, Oveisi A. Robust Control for Ride Comfort Improvement of an Active Suspension System considering Uncertain Driver's Biodynamics. Journal of Low Frequency Noise, Vibration and Active Control. 2014;33(3):317-339. doi:10.1260/0263-0923.33.3.17
- 3. Jibril, Mustefa (2020). H∞ and µ-synthesis Design of Quarter Car Active Suspension System. *International Journal of Scientific Research and Engineering Development* 3 (1):608-619. PhilArchive copy v1: https://philarchive.org/archive/JIBHAv1

# Thanks for listening